

Chapter 3:

*DYNAMICS OF PERFECT  
INCOMPRESSIBLE FLUIDS*

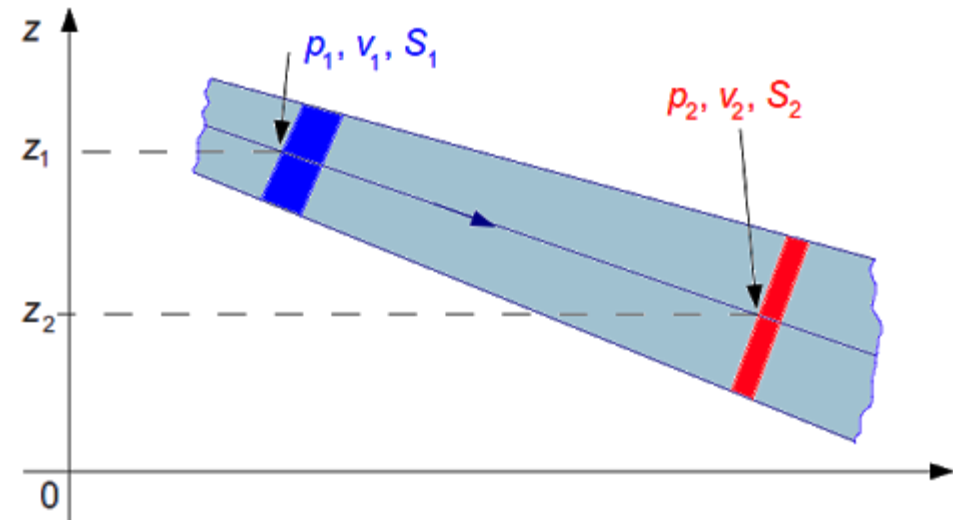
## 1/ INTRODUCTION

- Dynamics means fluids **in motion** . Unlike solids, moving parts of a fluid can move at different speeds.
- We use the equations that govern the dynamics of perfect incompressible fluids:

- The continuity equation (conservation of mass),
- Bernoulli's theorem (conservation of energy),
- Euler's theorem (conservation of momentum).

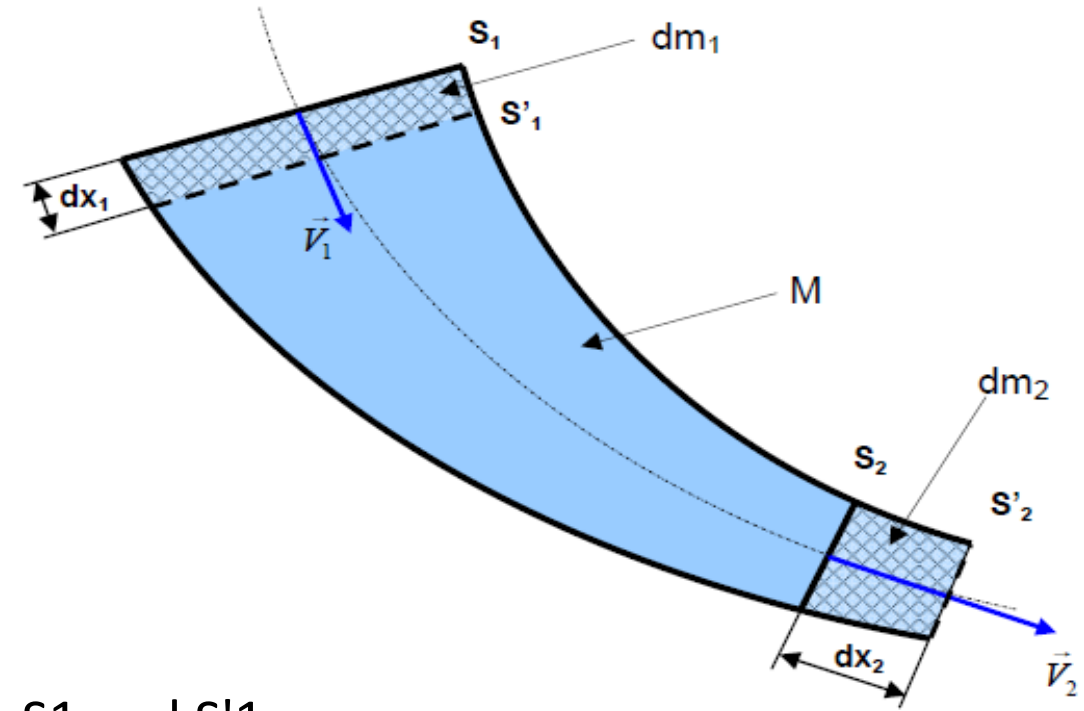
## 2/ PERMANENT FLOW

The flow of a fluid is said to be permanent if the field of velocity vectors of the fluid particles is constant at time  $t$ .



### 3/ CONTINUITY EQUATION

- Consider a vein of an incompressible fluid of density  $\rho$  animated by a permanent flow.
- **S1 and S2** respectively the input section and the fluid outlet section at time  $t$ ,
- **S'1 and S'2** respectively the sections entry and exit of the fluid at time  $t'=( t+dt )$ ,
- **$\vec{V}_1$  and  $\vec{V}_2$**  flow velocity vectors.
- **$dx_1$  and  $dx_2$**  : respectively the displacements sections S1 and S2 during the time interval  $dt$  ,
- **$dm_1$** : incoming elementary mass between sections S1 and S'1,
- **$dm_2$** : outgoing elementary mass between sections S2 and S'2,
- **$dV_1$**  : incoming elementary volume between sections S1 and S'1,
- **$dV_2$** : outgoing elementary volume between sections S2 and S'2,



We have:  $dV_1 = dV_2 \dots \dots \dots (1)$

Since we have:  $\rho = \frac{m}{V}$

$$\frac{dm_1}{\rho} = \frac{dm_2}{\rho} \text{ (because } \rho : \text{constante)}$$

**So:**  $dm_1 = dm_2$  (Conservation of mass)

According to relation (1):

$$dV_1 = dV_2$$

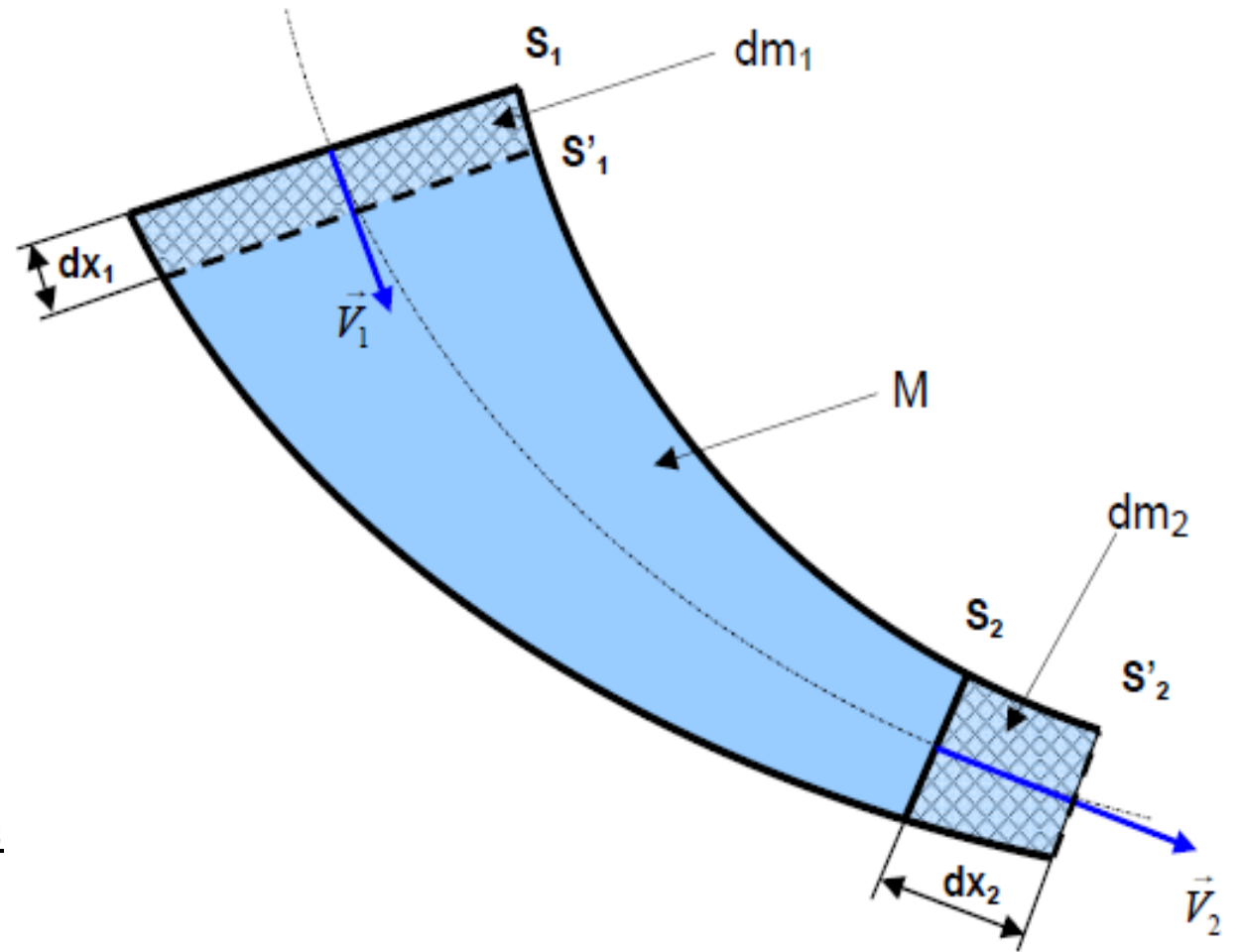
$$S_1 \cdot dx_1 = S_2 \cdot dx_2$$

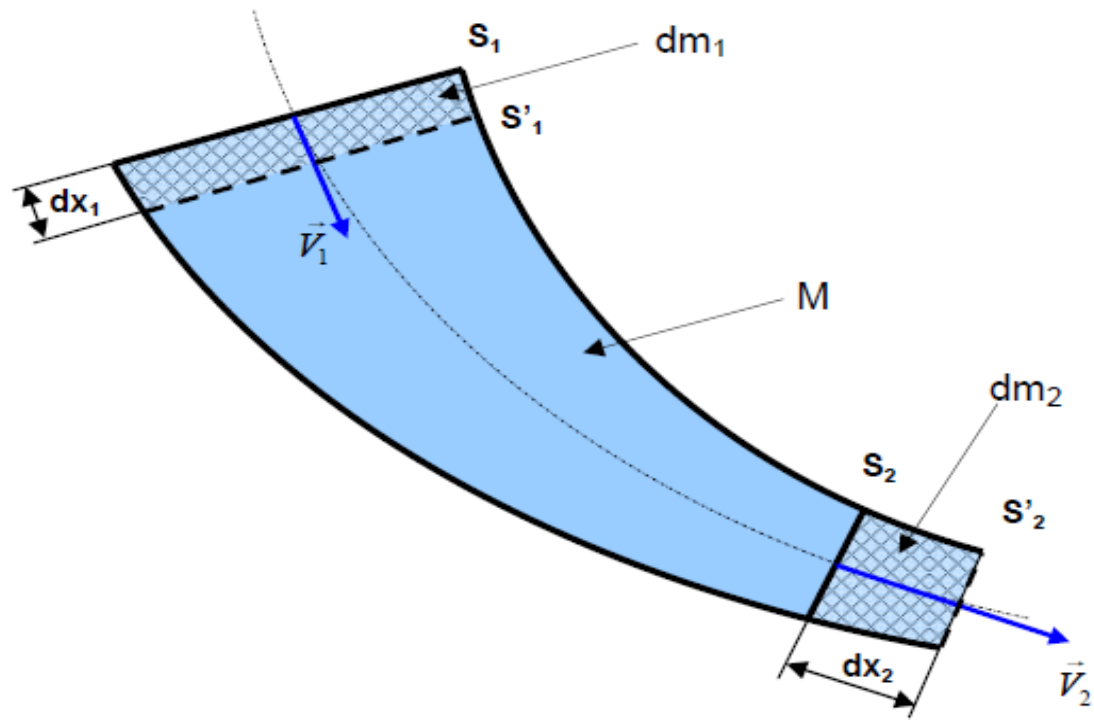
We divide the equation by dt :

$$S_1 \frac{dx_1}{dt} = S_2 \frac{dx_2}{dt}$$

So, we get:

$$S_1 \cdot V_1 = S_2 \cdot V_2 = \text{Constant}$$





$S_1 v_1 = S_2 v_2 = C^{ste}$  is the continuity equation

## 4/ CONCEPT OF THROUGHPUT

### 4.1/ Volume flow

Volume flow is the ratio of volume flow per unit time.

$$Q_V = \frac{dV}{dt}$$

noting that:  $dV = S \cdot dx$  we can also write that:

$$Q_V = \frac{dV}{dT} = \frac{Sdl}{dt} = Sv_{moy}, \quad [\mathbf{m^3/s}]$$

- **4.2/ Mass flow**

It is the mass of fluid per unit time:

$$Q_m = \frac{dm}{dT} = \frac{\rho dV}{dt} = \rho Q_V, \quad [\text{kg/s}] \quad (\text{Relation between } Q_m \text{ and } Q_v)$$

## 5/ BERNOULLI'S THEOREM – CASE OF A FLOW **WITHOUT EXCHANGE** OF WORK

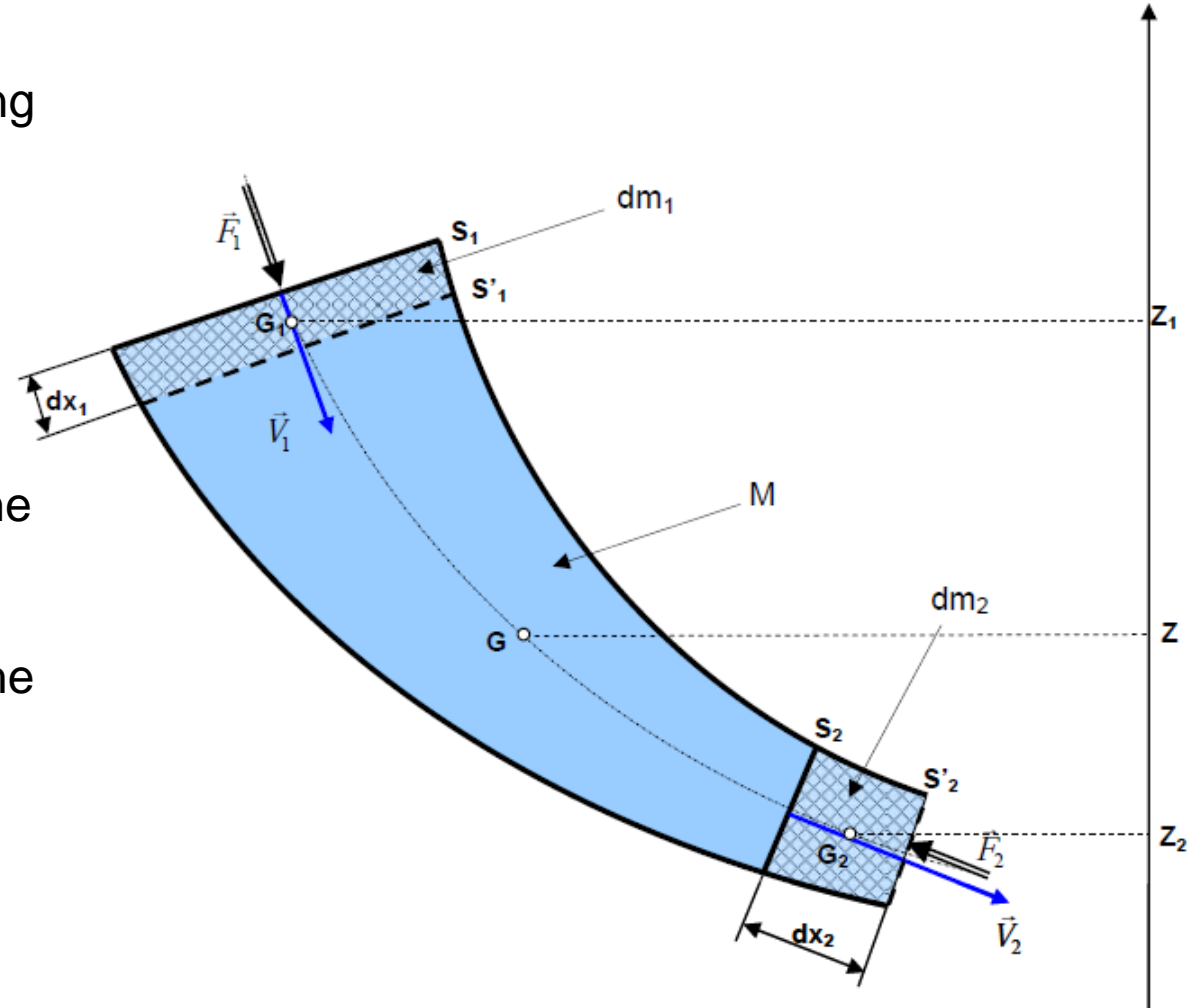
Let us take again the diagram of the fluid vein of paragraph 3 with the same notations and the following assumptions:

- The fluid is perfect and incompressible.
- The flow is permanent.
- The flow is in a perfectly smooth pipe .

We consider a vertical  $Z$  axis directed upwards.

We note  $Z_1$ ,  $Z_2$  and  $Z$  respectively the altitudes of the centers of gravity of the masses  $dm_1$ ,  $dm_2$  and  $M$ .

We denote by  $F_1$  and  $F_2$  respectively the norms of the pressure forces of the fluid acting at the level of the sections  $S_1$  and  $S_2$ .



At time t the mass fluid ( $dm_1 + M$ ) is between  $S_1$  and  $S_2$ . Its mechanical energy is:

$$E_{mec} = E_{pot} + E_{cin} = (dm_1 \cdot g \cdot Z_1 + MgZ) + \frac{1}{2} dm_1 \cdot V_1^2 + \int_{S_1}^{S_2} \frac{dm \cdot V^2}{2}$$

At time  $t' = (t + dt)$  the fluid of mass ( $M + dm_2$ ) is between  $S'_1$  and  $S'_2$ . Its mechanical energy is:

$$E'_{mec} = E'_{pot} + E'_{cin} = (MgZ + dm_2 \cdot g \cdot Z_2) + \int_{S'_1}^{S'_2} \frac{dm \cdot V^2}{2} + \frac{1}{2} dm_2 \cdot V_2^2$$

We apply the mechanical energy theorem to the fluid between t and t': "The variation of the mechanical energy is equal to the sum of the works of the external forces . »

$$E'_{mec} - E_{mec} = W_{\text{Pressure forces}} = F_1 \cdot dx_1 - F_2 \cdot dx_2 \Leftrightarrow E'_{mec} - E_{mec} = P_1 \cdot S_1 \cdot dx_1 - P_2 \cdot S_2 \cdot dx_2 = P_1 \cdot dV_1 - P_2 \cdot dV_2$$

by simplifying we obtain:  $dm_2 \cdot g \cdot Z_2 + \frac{1}{2} dm_2 \cdot V_2^2 - dm_1 \cdot g \cdot Z_1 - \frac{1}{2} dm_1 \cdot V_1^2 = \frac{P_1}{\rho_1} \cdot dm_1 - \frac{P_2}{\rho_2} \cdot dm_2$

By conservation of mass:  $dm_1 = dm_2 = dm$  and since the fluid is

incompressible:  $\rho_1 = \rho_2 = \rho$ , We arrive at Bernoulli's equation:

$$\boxed{\frac{V_2^2 - V_1^2}{2} + \frac{P_2 - P_1}{\rho} + g(Z_2 - Z_1) = 0}$$

The unit of each term of the relation (4) is the joule per kilogram (J/kg)

According to relation (4), we can then write:

$$\frac{V_2^2}{2} + \frac{P_2}{\rho} + g \cdot z_2 = \frac{V_1^2}{2} + \frac{P_1}{\rho} + g \cdot z_1$$

## 6/ BERNOULLI'S THEOREM – CASE OF A FLOW **WITH** EXCHANGE OF WORK

It is further assumed that a hydraulic machine is placed between the sections  $S_1$  and  $S_2$ . This machine is characterized by a net power  $P_{net}$  exchanged with the fluid, a power on the shaft  $P_a$  and a certain efficiency  $\eta$ . This machine can be either a turbine or a pump.

- In the case of a pump: the efficiency is given by the following expression:

$$\eta = \frac{P_{net}}{P_a}$$

- In the case of a turbine: the efficiency is given by the following expression:

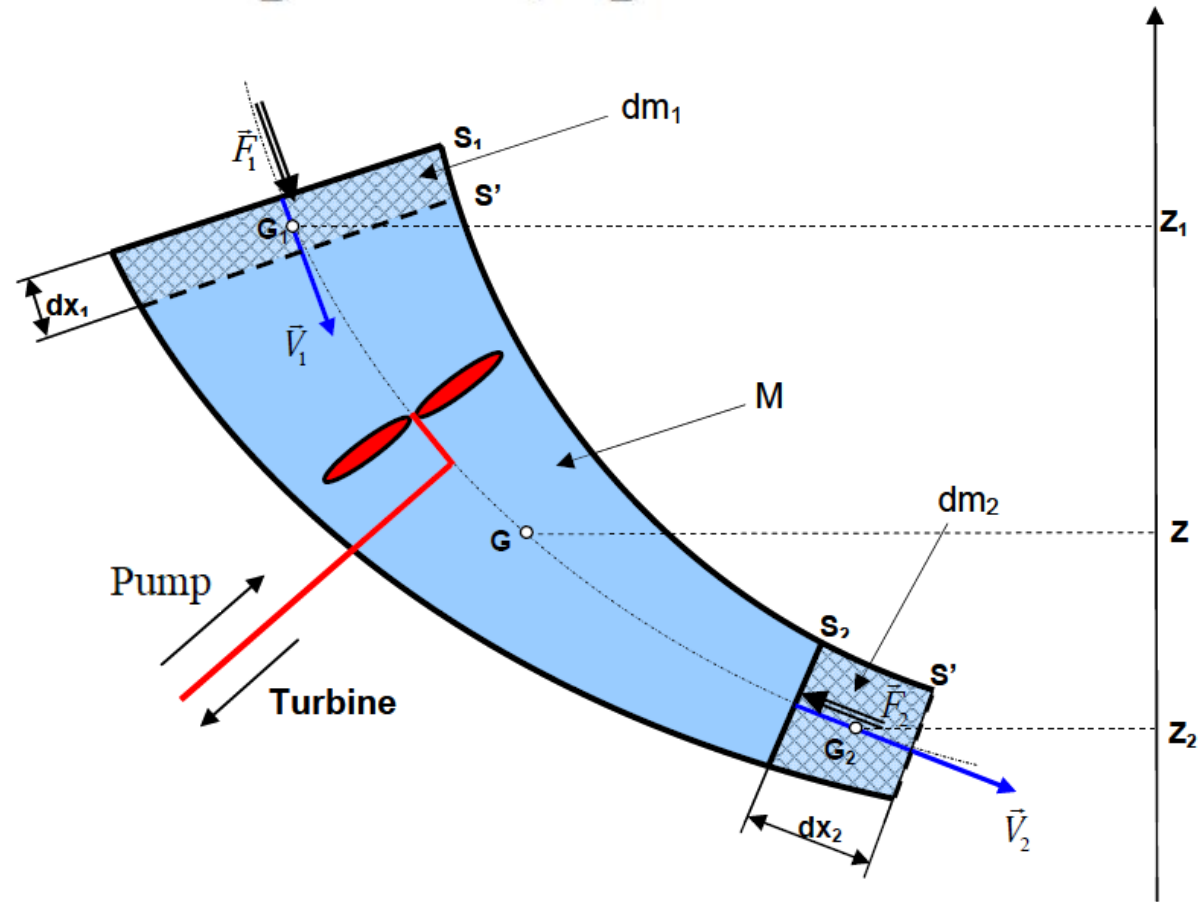
$$\eta = \frac{P_a}{P_{net}}$$

Between times  $t$  and  $t'=(t+dt)$ , the fluid exchanged net work  $W_{net} = P_{net} \cdot dt$  with the hydraulic machine.  $W_{net}$  is assumed to be positive if it is a pump and negative if it is a turbine.

We denote by  $F_1$  and  $F_2$  respectively the standards of the pressure forces of the fluid acting at sections  $S_1$  and  $S_2$ .

At time  $t$  the mass fluid ( $dm_1 + M$ ) is between  $S_1$  and  $S_2$ . Its mechanical energy :

$$E_{mec} = E_{pot} + E_{cin} = (dm_1 \cdot g \cdot Z_1 + MgZ) + \frac{1}{2} dm_1 \cdot V_1^2 + \int_{S_1}^{S_2} \frac{dm \cdot V^2}{2}$$



At time  $t'=(t+dt)$  the mass fluid  $(M+dm_2)$  is between  $S'_1$  and  $S'_2$ . Her

$$\text{mechanical energy is: } E'_{mec} = E'_{pot} + E'_{cin} = (MgZ + dm_2 \cdot g \cdot Z_2) + \int_{S'_1}^{S'_2} \frac{dm \cdot V^2}{2} + \frac{1}{2} dm_2 \cdot V_2^2$$

We apply the mechanical energy theorem to the fluid between  $t$  and  $t'$ : «

The variation of the mechanical energy is equal to the sum of the works of

the external forces. »>, considering this time the work of the hydraulic machine

$$E'_{mec} - E_{mec} = F_1 \cdot dx_1 - F_2 \cdot dx_2 + P_{net} \cdot dt$$

$$E'_{mec} - E_{mec} = P_1 \cdot S_1 \cdot dx_1 - P_2 \cdot S_2 \cdot dx_2 + P_{net} \cdot dt = P_1 \cdot dV_1 - P_2 \cdot dV_2 + P_{net} \cdot dt \text{ by simplifying we will have:}$$

$$dm_2 \cdot g \cdot Z_2 + \frac{1}{2} dm_2 \cdot V_2^2 - dm_1 \cdot g \cdot Z_1 - \frac{1}{2} \cdot dm_1 \cdot V_1^2 = \frac{P_1}{\rho_1} \cdot dm_1 - \frac{P_2}{\rho_2} \cdot dm_2 + P_{net} \cdot dt \quad \text{Per conservation}$$

mass:  $dm_1 = dm_2 = dm$  and since the fluid is incompressible:  $\rho_1 = \rho_2 = \rho$ ,

we arrive at Bernoulli's equation

$$\boxed{\frac{V_2^2 - V_1^2}{2} + \frac{P_2 - P_1}{\rho} + g(Z_2 - Z_1) = \frac{P_{net}}{\rho}} \quad (5)$$

- Momentum ( EULER'S THEOREM )

This theorem makes it possible to determine the forces exerted by the moving fluid on the objects which surround them .

- Euler's theorem results from the application of the momentum theorem to the flow of a fluid:

$$\sum \vec{F}_{ext} = \frac{d\vec{P}}{dt} \quad \text{with } \vec{P} = m\vec{V}_G : \text{ momentum.}$$

$$\sum \vec{F}_{ext} = q_m (\vec{V}_2 - \vec{V}_1)$$